

## **Bottom Outlet Hydraulics**

Benjamin Hohermuth, Lukas Schmocker, Robert Boes SCCER-SoE Annual Conference 14<sup>th</sup> of September 2018, Horw

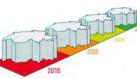




## **Motivation**

Adapt hydraulic structures to meet future demands

Swiss Energy Strategy 2050



+ 3.1 TWh/a of flexible peak and winter energy in 2050

→ Dam heightening

Climate Change



Glacier/permafrost retreat exposes instable hillslopes

- → Increased risk of impulse waves
- Increased reservoir sedimentation in periglacial environments

#### $\rightarrow$ Increased load on outlet structures





Dam heightening at Luzzone 1998

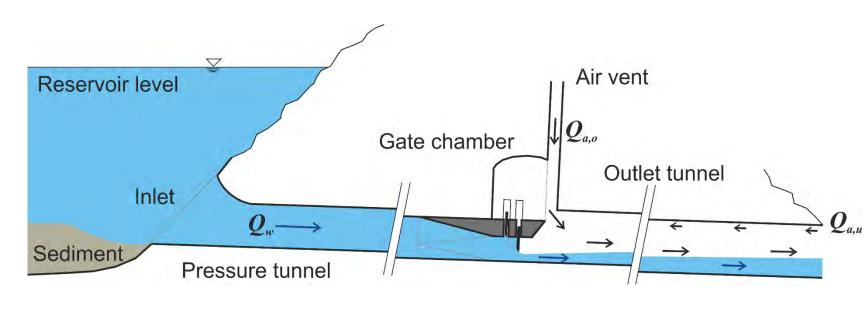


Reservoir sedimentation at Griessee



## **Bottom Outlets: Key Safety Devices**

Purposes & Challenges



### Purposes:

- Control of reservoir level
- Sediment flushing
- Residual flow release
- Flood discharge

### Challenges:

- Cavitation
- Gate vibration
- Flow choking / slugs

### $\rightarrow$ «sufficient» aeration is crucial for a safe operation

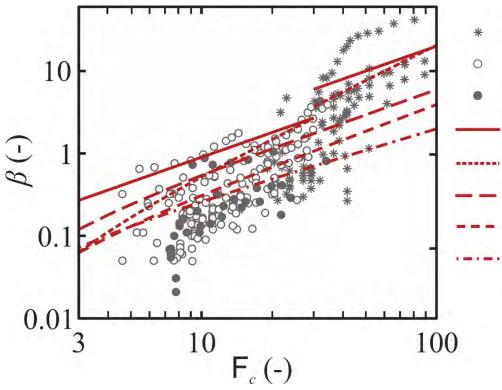


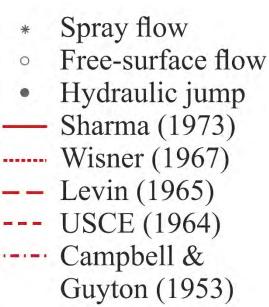


#### EHzürich

## State of Knowledge

Air demand  $\beta = Q_{a,o}/Q_w$ 





### Goal

Improve design guidelines by including the effects of

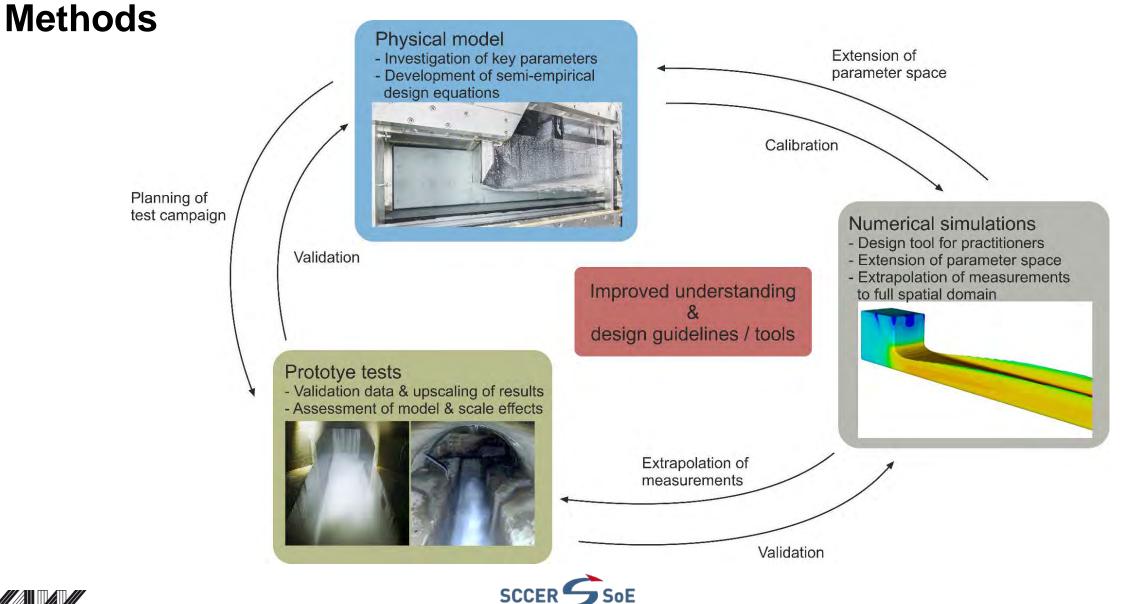
- Air vent
- Tunnel length
- Tunnel slope





#### EHzürich

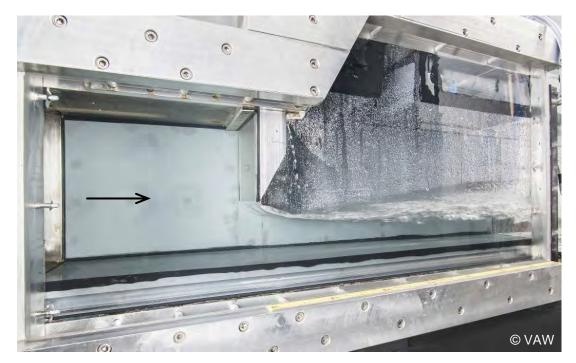
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### **Physical Model** Model scale $\lambda \approx 10$



Max. length L = 20.6 m Max. energy head  $H_E = 30$  m w.c. Max. discharge  $Q_w = 600$  l/s



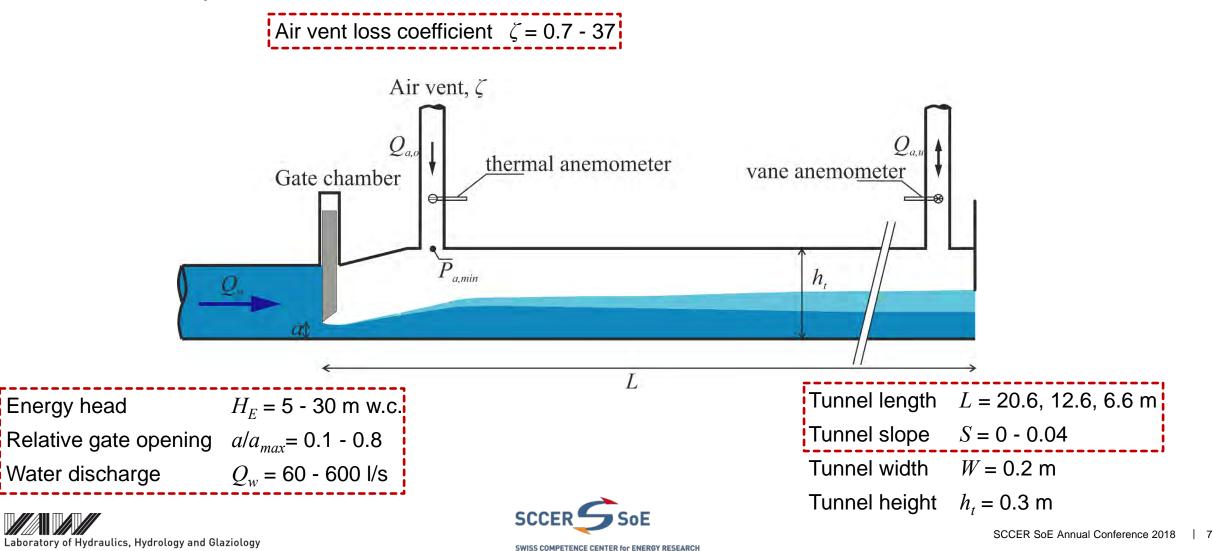
Max. flow velocity @ vena contracta  $\approx 24$  m/s Max. gate opening  $a_{max} = 0.25$  m Tunnel width W = 0.2 m, tunnel height  $h_t = 0.3$  m





## **Physical Model**

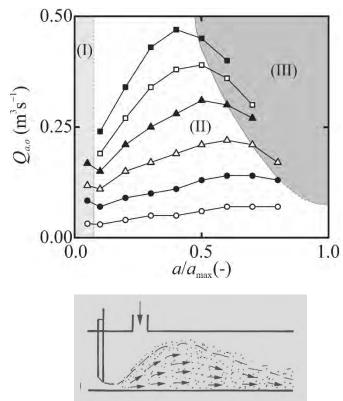
General setup



SUPPLY of ELECTRICITY

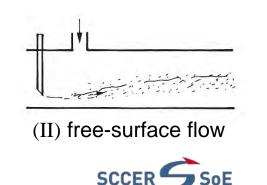
## **Model Tests: Air Demand**

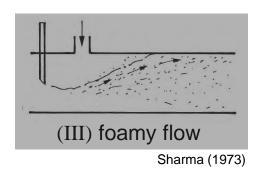
Effect of flow pattern,  $a/a_{max}$  and  $H_E$ 

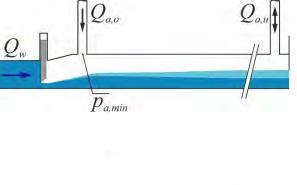


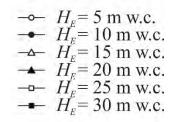
(I) spray flow

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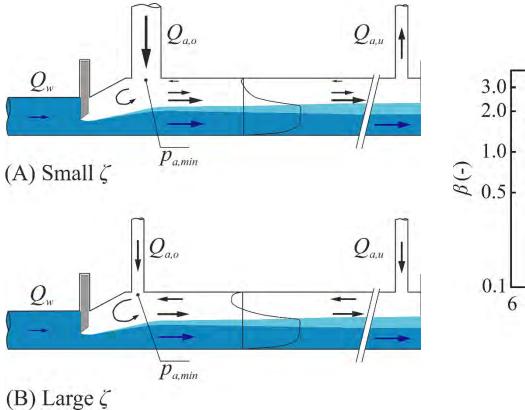


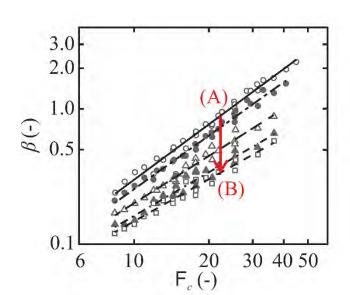


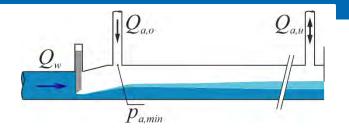


## **Model Tests: Air Demand**

Effect of air vent loss coefficient  $\zeta$ 







 $\zeta = 0.7$   $\zeta = 2.7$   $\zeta = 8.9$   $\zeta = 19$   $\zeta = 28$ 





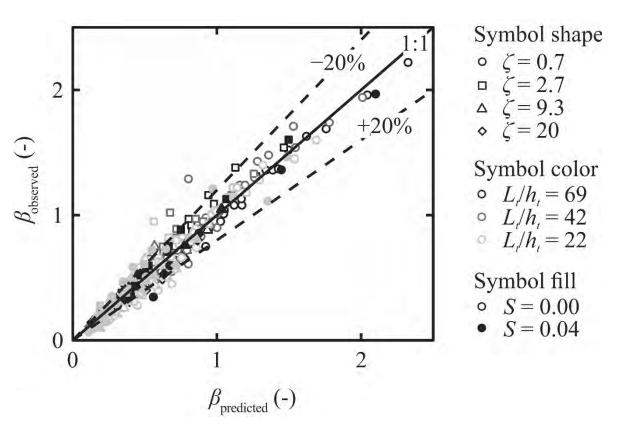
## Model Tests: Air Demand

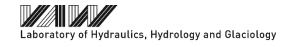
**Design Equation** 

 $\beta = 0.007 \,\mathsf{F}_{c}^{1.20} \zeta^{-0.25} (L_t/h_t)^{0.26} (1+S)^{-0.92}$ 

### Limitations

- Free surface flow
- No profile transition
- $8 \le F_c \le 45$
- 0.7 ≤ ζ ≤ 20
- $22 \le L_t/h_t \le 69$
- 0 ≤ *S* ≤ 0.04







## **Air Demand**

Comparison to prototype data

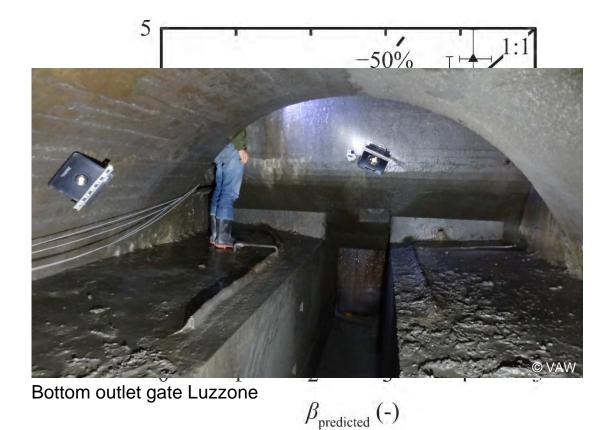
### Data from literature

- Curnera (Lier & Volkart 1994)
- Mauvoisin (Schilling 1963)
- Norfork (USACE 1954)  $\rightarrow$  SF = 1.2
- Panix (Volkart & Speerli 1994) → SF ~ 3

### Safety factors (SF) for practical applications

- No profile transition
- Smooth profile transition
- Abrupt profile transition
   (Levin 1965, Sharma 1973)







 $\rightarrow$  SF = 2

SF = 1.2

SF = 3 to 4

SF = 2

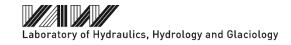
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### **Prototype tests at Malvaglia and Luzzone**

In cooperation with Ofible, financially supported by Lombardi Engineering Foundation







## **Conclusions & Outlook**

- Increased process understanding from physical model tests
- Improved design equation
- First validation of model results with prototype data from literature

- More high-quality prototype data needed to further reduce uncertainty for upscaling & practical applications
- Numerical design tools require further development







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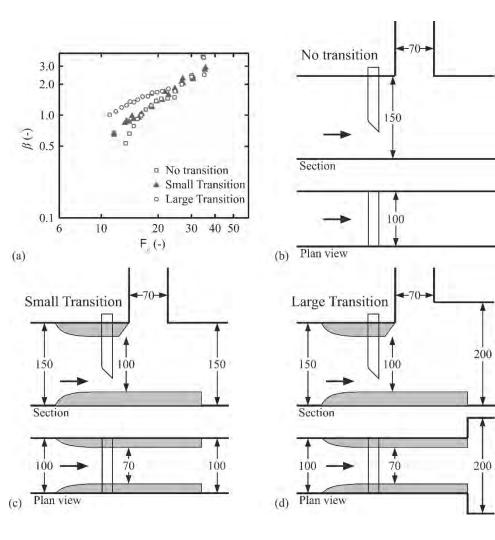




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### **Air demand: Effect of Profile Transition**

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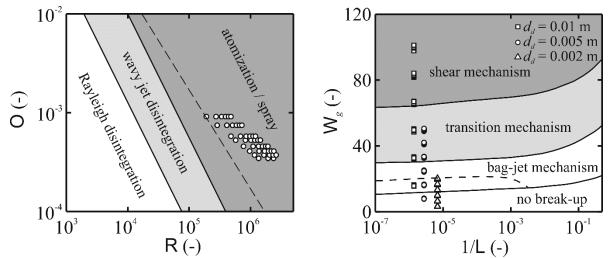
## Model Tests: Slug flow

Effect of air vent loss coefficient  $\zeta$ 



### **Physical model** Scale effects

- 1) Air entrainment: Limiting Weber number W > 170 (Skripalle 1994)
- 2) Jet break-up: Same regime



$$W = (u^2 \rho L / \sigma)^{0.5}$$
  

$$R = uL/v$$
  

$$O = W/R$$
  

$$L = (\rho \sigma L)/\eta^2$$

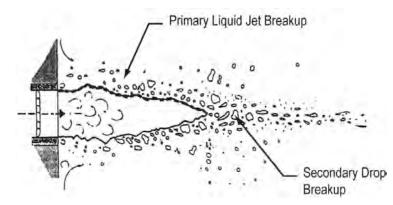


Illustration of jet break-up (Trinh 2007)

L = reference length u = reference velocity  $\eta$  = dynamic viscosity v = kinematic viscosity  $\rho$  = density  $\sigma$  = surface tension coeff.

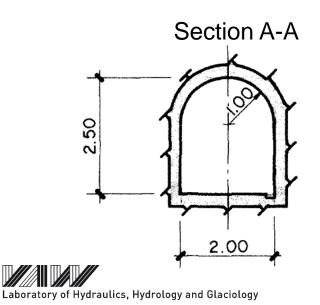
(left) Primary jet break-up (Ohnesorge 1936) (right) Secondary jet break-up (Krezeczkowski 1980)

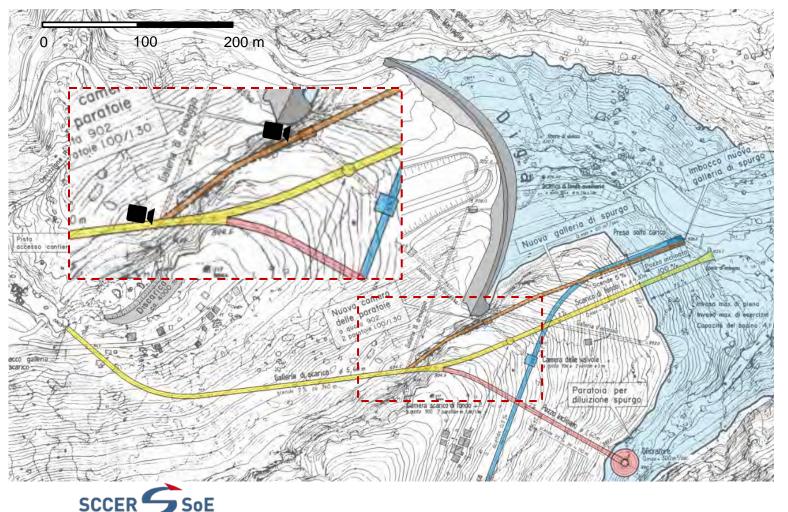
## **Prototype Tests HPP Blenio**

Luzzone and Malvaglia dam

### New bottom outlet

- Scale compared to phys. Model  $\lambda \approx 10$
- *H<sub>E</sub>* = 82 m w.c.
- *L* = 82 m



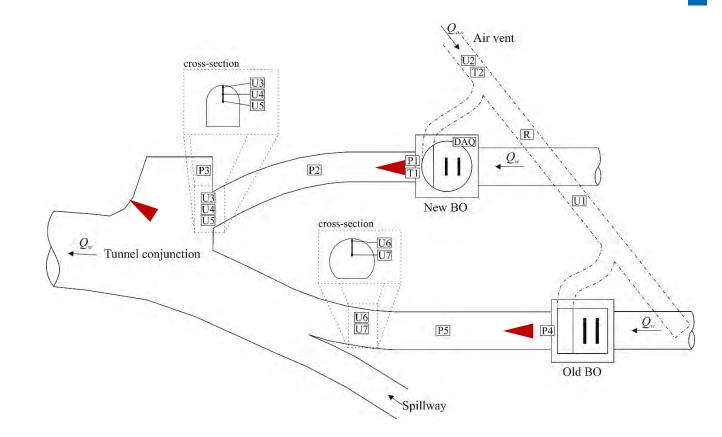


# Malvaglia Test Site

Instrumentation

Measurement of:

- Air vent discharge  $Q_{a,o}$
- Air flow from d/s  $Q_{a,u}$
- Air vent loss coefficient  $\zeta$ 
  - Air pressure  $p_a$
  - Air temperature  $T_a$

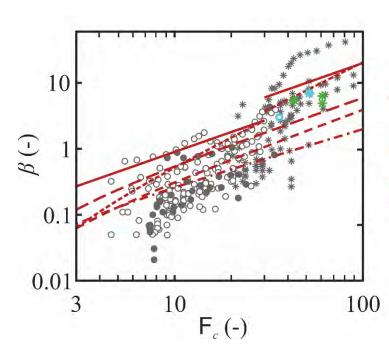


U1-2: Vane anemometers 0-120 m/s U3-7: Vane anemometers -60-60 m/s P1-5: Absolute pressure sensors 700-1200 mbar T1-2: Air temperature sensors -30-40°C R: Flow direction sensor 0-360°

## **Prototype Tests: Preliminary Results**

New Bottom Outlet Malvaglia

- Influence of flow pattern
- Influence of F<sub>c</sub>
- Influence of  $\zeta$



- \* Spray flow
- Free surface flow (fsf)
- Hydraulic jump
- Sharma (1973)
- ----- Wisner (1967)
- -- Levin (1965)
- --- USCE (1964)
- ---- Campbell & Guyton (1953)
- Malvalglia old, spray
- Malvaglia new, spray
- Malvaglia new, fsf



